

Design and Analysis of Piston

Hardik Nayak¹, Mistry Yash², Patel Bhavik³, Patel Dev⁴, Rabari Sajan⁵, Patel Shrikunj⁶

Assistant Professor, Mechanical Engineering Department, R.N.G. Patel Institute of Technology, Bardoli, Gujarat, India¹

B.Tech Student, Mechanical Engineering Department, R.N.G. Patel Institute of Technology, Bardoli, Gujarat, India²

B.Tech Student, Mechanical Engineering Department, R.N.G. Patel Institute of Technology, Bardoli, Gujarat, India³

B.Tech Student, Mechanical Engineering Department, R.N.G. Patel Institute of Technology, Bardoli, Gujarat, India⁴

B.Tech Student, Mechanical Engineering Department, R.N.G. Patel Institute of Technology, Bardoli, Gujarat, India⁵

B.Tech Student, Mechanical Engineering Department, R.N.G. Patel Institute of Technology, Bardoli, Gujarat, India⁶

Submitted: 05-06-2021

Revised: 18-06-2021

Accepted: 20-06-2021

ABSTRACT: The Piston is part of the engine which converts heat and pressure liberated by fuel combustion into mechanical work. The engine piston is the most complex automotive component. This article describes the design and analysis of pistons in different material (cast iron, gray cast iron and aluminum alloys) and different conditions. Our purpose for this article is to check that which material is best for piston under different conditions. The work is used to calculate the distribution of stresses, temperature and shear stress on the piston surface. Mostly due to mechanical and thermal stress, piston fail. we applied temperature 720-degree Celsius on the piston. Ansys provide a simulation solution. Design of piston is carried out using solid works and different conditions perform in Ansys.

I. INTRODUCTION

A disc or small cylinder that fits tightly inside a tube and travels up and down against a liquid or gas, used to generate motion in an internal combustion engine or to impart motion in a pump. A piston can be used in reciprocating engines, reciprocating pumps, gas compressors, hydraulic cylinders, and pneumatic cylinders, among other things.

Type of piston head:

1. **Flat piston head:** Flat piston has a flat top. These pistons are typically used in mass

produced engines. They are easy to manufacture and low cost of engines.

Dish/bowl piston head: The piston has a bowl or dish shaped top. It is used to reduce compression ratio because it adds volume to the chamber. It can be used in turbocharged or supercharged engine to help avoid detonation.

Dome Pistons head: These are the polar opposite of the dish pistons in concept, since they bubble in the center like a stadium's roof. This is achieved to maximize the usable surface area on the piston's top. Less compression means more surface area.

Type of piston

Trunk piston: composite construction of these pistons consists of a thin section. The titanium alloy steel piston crown and aluminum alloy skirt. These pistons are made of thin, solid, and rigid materials that can withstand high temperatures and corrosion.

Cross head piston: Piston crown, piston skirt, and piston rod (used in massive two-stroke engines) are attached to the crosshead, which transfers side thrust to the engine structure.

Slipper piston: A slipper piston is a piston for a petrol engine that has been reduced in size and weight as much as possible. They are reduced to the piston crown, support for the piston rings, and just enough of the piston skirt remaining to leave two lands so as to stop the piston rocking in the bore.

Deflector piston: Deflector pistons are used in two-stroke engines with crankcase compression, where the gas flow within the cylinder must be carefully directed in order to provide efficient scavenging. With cross scavenging, the transfer (inlet to the cylinder) and exhaust ports are on directly facing sides of the cylinder wall. To prevent the incoming mixture passing straight across from one port to the other, the piston has a raised rib on its crown.

Functions of piston:

- to transmit gas forces to the crank shaft through the connecting rod, to seal the combustion chamber against gas leakage to the crankcase - in combination with the piston rings - and prevent oil penetration from the crankcase into the combustion chamber.
- the chamber of combustion.
- absorbing combustion heat and dissipating it to the cylinder liner and cooling oil.

Piston material:

Gray cast iron: It is named after the gray color of the fracture it forms, which is due to the presence of Graphite. It is used for housings where the stiffness of the component is more important than its tensile strength. Such as internal combustion engine cylinder blocks.

Aluminum: Aluminum is a metal that is lightweight, solid, flexible, non-corrosive, and always recyclable. In the transportation, manufacturing, packaging, and electrical industries, aluminum is commonly used.

Cast iron: Cast iron is an iron alloy containing 2 to 4% carbon, as well as varying percentages of silicon and manganese, as well as amounts of sulfur and phosphorus. It is produced in a blast furnace by reducing iron ore.

Design Procedure

Design parameter for piston:

Cylinder Bore (D) = 110 mm

Piston Stroke (L) = 120 mm = 0.12 m

Permissible Tensile Stress for the material of the piston = 35 N/mm² Maximum Pressure (P) = 5 N/mm²

Constant Heat Supplied to Engine (C) = 0.05 (Constant)

Higher Calorific Value of fuel (HCV) = 42,000 KJ/Kg

Mass of the fuel used in kg per brake power per second = 0.15 kg Indicated mean effective pressure (pm) = 0.75 N/mm²

Speed (N) = 200 rpm

Mechanical efficiency = 80%

0.8 Thermal Conductivity = 46.6 W/mc

Thickness of Piston head (th): $Th = 0.433 * D * \sqrt{\text{root of } (P_{max}/ft)}$ **Th = 18 mm**

Heat through the piston: $H = C + HCV + M + BP$

$$I_p = \frac{P_m L A N}{60}$$

$$0.75 * 0.73 * 9504 * 1000$$

$$=$$

$$60$$

$$= 14256$$

$$= 14.26 \text{ KW BWP} = IP * NM$$

$$= 14.26 * 0.8$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

$$= 11.40 \text{ KW}$$

Thickness of Piston Head On The Basis Of Head Displacement: $Th = \frac{H}{12.56 * k * (Tc - Te)}$

$12.56 * k * (Tc - Te)$

$$= \frac{996}{12.56 * 46.6 * 220}$$

$$= 7.73 * 10^{-3} \text{ m}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

$$= 7.73 \text{ mm}$$

Width of the rringland $\phi_2 = 0.75 \phi_2$ to ϕ_2
 $= 0.75 * 4$ to 4
 $= 3$ to 4
 $b_2 = 3.5$ mm
 Free ring (G) = $3 \phi_1$ To $4 \phi_1$

$= 3 * 4.5$ to $4 * 4.5$
 $= 13.5$ to 18
 $= 16$ mm

Maximum Thickness of Barrel ϕ_3

Radial depth of piston ring groove (b) $b = \phi_1 + 0.4$
 $= 4.5 + 0.4$

b = 4.9 mm

$\phi_3 = 0.03D + b + 4.5$
 $= (0.03 * 110) + 4.9 + 4.5$
 $\phi_3 = 10.1$ mm

Piston wall thickness towards the open end:

$\phi_4 = 0.25 + 3 + 0$ to $0.35 + 3$

$= 0.25 * 12.7$ to $0.35 * 12.7$
 $= 3.17$ to 4.44 mm
 $\phi_4 = 4$ mm

Piston pin:

$\phi_1 = 0.45 * D$
 $= 0.45 * 110$
 $= 19.5$ mm
 $\phi_0 = \phi_1$ Pressure
 $= 20$ mm Load on pin begins pressure
 $= \phi_0 * \phi_0 * \phi_1$

$= 20 * \phi_0 * 495$
 $= 990$ mm

Maximum load on piston due to gas Pressure (fgmax)

$= (\pi/4) * D^2 * P$

[p = maximum gas pressure = 5 N/MM²]
 $= (\pi/4) * (110)^2 * 5$
 $= 47516.59$ N

Fgmax = 1 mm

$47516.59 = 990 * \phi_0$

$\phi_0 = 47.99$

Do = 48 mm

Inside diameter (ϕ_1) = $0.6 * do$

$= 0.6 * 48$

$= 28.8$ mm

$\phi_1 = 29$ mm Piston diameter

Maximum side thrust on cylinder due to gas

pressure (p) $R = \mu * (\pi/4) D^2 * P$

$= 47516.59$ N

Bearing Pressure on the piston barrel (ϕ_0) $R = \phi_0 * D$

$* 404751.65 = 0.4 * 110 * 40$

$\phi_0 = 107.99$ mm

= 110 mm Length of piston:

L = length of

skirt (ϕ_1) + length of

the ring section + top barrel

$= \phi_1 + [4 + 2 + 3 \phi_2] + \phi_1$

$= 110 + [4 * 4 + 3 * 3.5] + 20$

$= 156.5$ mm

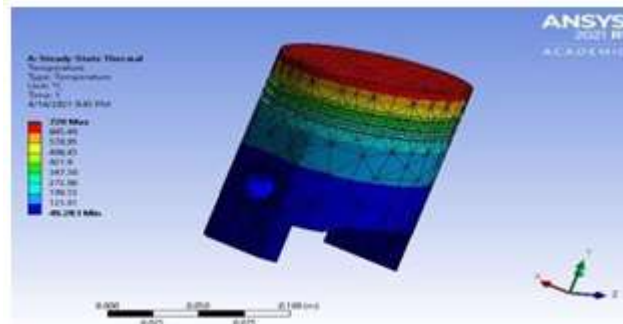
$= 156$ mm

Design of piston and important parameter:

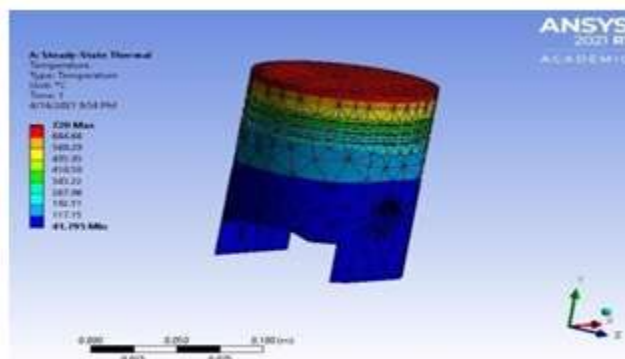
Parameter	Dimension	Unit
Thickness of piston head (t_h)	18	mm
Radial thickness (T1)	4.5	mm
Axial thickness (T2)	4	mm
Width of the top land (b1)	20	mm
Width of other land (b2)	3.5	mm
Inside diameter of piston (d_1)	29	mm
Length of the piston (L)	159	mm
Diameter of piston (D)	110	mm



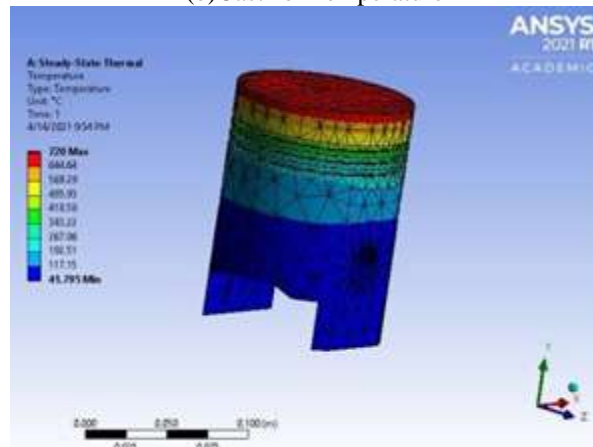
Design Of Piston



1(a)aluminumtemperature



1(b)CastIronTemperature

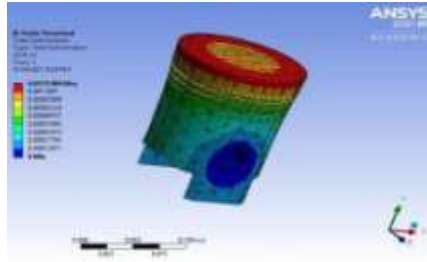


1(c) Gray Cast Iron Temperature

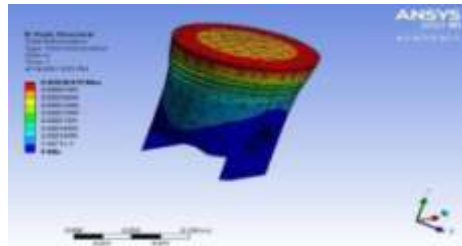
Temperature of all material for piston

No	Time(s)	Parameter	Aluminum	Cast iron	Gray cast iron
1	1	Minimum	174.34	49.283	41.795
2	1	Maximum	720	720	720
3	1	Average	416.64	295.2	206.31

Deformation of Piston In Ansys:



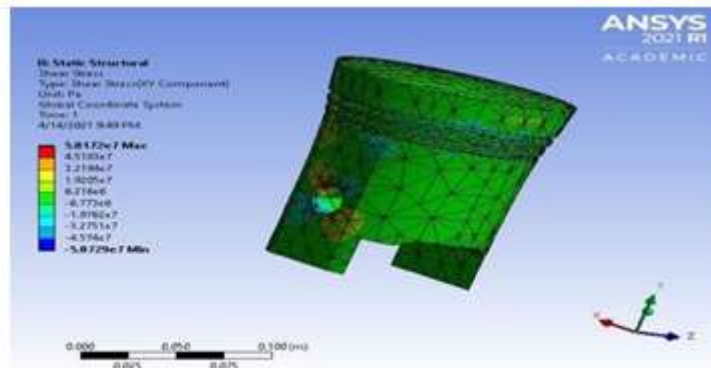
2(a)castirondeformation



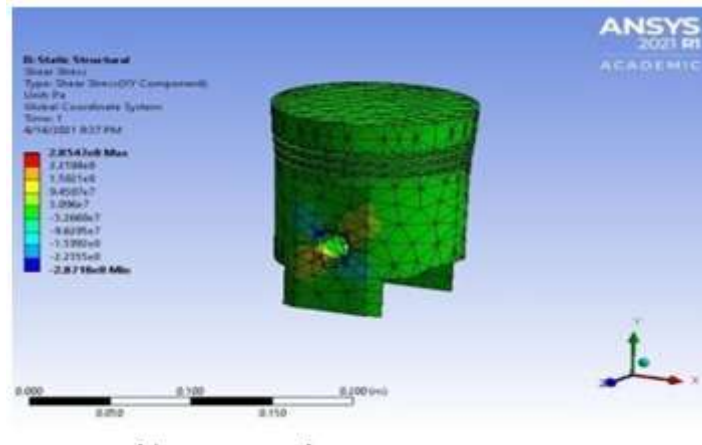
2(b)Aluminiumdeformation

1(c)Gray cast iron deformation **Deformationtablefor allmaterial**

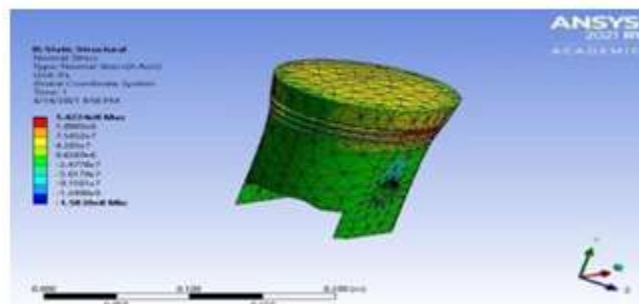
No	Time(s)	Parameter	Aluminum	Castiron	Gray castiron
1	1	Minimum	-2.8718e+008	-5.8729e+007	-5.5517e+007
2	1	Maximum	2.8547+008	5.8172e+007	5.6017e+007
3	1	Average	67371	41381	-73793



3(a)aluminumShearStress



3(b)CastIronShearStress



3(C)GrayCastIronShearStress

shear stress of all material for piston

N o	Time(s)	Parameter	Aluminum	Castiron	Gray castiron
1	1	Minimum	0	0	0
2	1	Maximum	1.2484e-003	5.4719e-004	4.97479e-004
3	1	Average	5.3918e-004	1.8659e-004	1.0789e-004

II. CONCLUSION:

The fundamental concepts and design methods concerned with four stroke diesel engine have been studied in this paper. The results found by the use of the machine design book method are nearly equal to the actual dimensions. Hence the design for the piston made in SolidWorks and analysis in Ansys software for different materials: cast iron, aluminum, and gray cast iron are studied in different conditions: total deformation, under temperature, and shear stress comparison for this. At the end of the project we found that gray cast iron is the best material for piston, but its manufacturing cost and weight of piston increase while a

luminum is the cheapest material but it cannot withstand high temperature, but we can use the aluminum alloy to solve some amount of this problem.

REFERENCES

- [1] Analysis of aluminum alloy piston” by B.A.Devan1, G. Ravinder Reddy2 International Journal of Emerging Trends Ghosh, Ashok Kumar Malik, Theory of Mechanism and Machines, third Edition, Affiliated press pvt limited New Delhi 1998.
- [2] Shigley Computer Aided Design and Analysis of Piston Mechanism of Four Stroke S.I. Engine.
- [3] The Science and Technology of Materials in Aut

- omotiveEngines
- [4] Amitabha, Joseph Edward, Theory of Machines and Mechanisms, Tata McGraw Hill, New York, 2003.
 - [5] Khurmi, R.S. and Gupta, J.K., A Textbook of Theory of Machine, 4th Edition, Eurasia Publishing House (Pvt.), Ltd, New Delhi, 2003
 - [6] B.A.Devan1, G. Ravinder “Thermal analysis of aluminum alloy piston” International Journal of Emerging Trends in Engineering Research (IJETER), vol.3 no6, P P:511-515(2015).
 - [7] “Thermal in Engineering Research (IJETER), vol. 3 no6, ISSN 2347-3983. Pages: 511 -515 (2015) 2. Ajay Raj Singh, Dr. Pushendra Kumar Sharma, “Design , Analysis and Optimization of Three Aluminum Piston Alloys Using FEA” Int. Journal of Engineering Research and Applications, ISSN : 2248-9622, Vol. 4, Issue 1 Version 3, January 2014, pp.94-102
 - [8] R.S.Khurmi, J.K.Gupta, A Text Book of Machine Design, S.Chand & Co., pp.1132-1144, 2004.
 - [9] Shuoguo Zhao, Design the piston of Internal Combustion Engine by PRO/ENGINEER, EMEIT-2012