

# Design and Static Structural Analysis of Leaf Spring by Using Composite Materials for Light Vehicles

<sup>1</sup>D.G. Kantharaj, <sup>2</sup>B. Anjaneyulu

<sup>1</sup>PG Student, Department of Mechanical Engineering, Gates Institute of Technology, Gooty-515401, A.P, India.

<sup>2</sup>Associate Professor & P. G Coordinator, Department of Mechanical Engineering, Gates Institute of Technology, Gooty-515401, A.P, India.

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**ABSTRACT:** In now a day the fuel efficiency and emission gas regulation of automobiles are two important issues. To fulfil this problem the automobile industries are trying to make new vehicle which can provide high efficiency with low cost. The best way to increase the fuel efficiency is to reduce the weight of the automobile. The weight reduction can be achieved primarily by the introduction of better material, design optimization and better manufacturing processes. The achievement of weight reduction with adequate improvement of mechanical properties has made composite a very good replacement material for conventional steel. To reduce vehicle weight, three techniques have been studied rationalizing the body structure, utilizing lightweight materials for parts and decreasing the size of the vehicles. In this approach by introducing composite materials into automobile industries, which is having low cost, high strength to weight ratio and excellent corrosive resistance can fulfil the requirement.

The automobile vehicles have number of parts which can be able to replace by composite material, but due to the improvement of mechanical properties of composite material. It has more elastic strength and high strength to weight ratio has compared with those of steel material. So, out of many components one of the components of automobile, the leaf spring which use for carried out the whole weight of the vehicle is best option for replacement of steel material by composite materials. Leaf springs are used in suspension systems. The automobile industry has shown increased interest in the replacement of steel leaf spring with composite leaf spring due to high strength to weight ratio.

Composite materials are one of the material families which are attracting researchers and being solutions of such issue. In this paper we describe design and

In this project we demonstrate the design and analysis of a Mahindra Jeep Commander 650 Di leaf spring is considered which is made up of coir

reinforced polyester matrix composite and fibre reinforced polymer (S-glass/epoxy). The main objective of this project is to reduce the weight of an automobile by replacing its steel leaf spring with composite leaf spring when the load applied is constant and at the same deflection compare its relative merits and demerits. The modelling of leaf-springs is done using NX -10.0 and analysis is done using ANSYS 18.1.

**KEYWORDS:**Key words: Stiffness, Composite Leaf Spring, E-Glass/Epoxy, ANSYS 18.1, NX10.0 – CAD.

## I. INTRODUCTION

suspension system is one having springs and other devices that insulate the chassis of a vehicle from shocks transmitted through the wheels.

The main components of the suspension system are:

- Struts
- Shock absorbers
- Springs
- Tires

The automobile chassis is mounted by the axles, not directly but through some form of springs. This is done to isolate the vehicle body from the road shocks which may be in the form of bounce, pitch, roll or sway. These tendencies give rise to an uncomfortable ride and also cause additional stress in the automobile frame and body. All the part performs the function of isolating the automobile from the road shocks are collectively called a suspension system. It also includes the spring device and various mountings. A suspension system consists of a spring and a damper. The energy of road shock causes the spring to oscillate. These oscillations are restricted to a reasonable level by the damper, which is more commonly called a shock absorber. A spring is defined as an elastic body, whose function is to distort when loaded and to recover its original shape when the load is removed. The different types of springs are:

1. Helical springs
2. Conical and volute springs
3. Torsion and spiral springs
4. Leaf springs
5. Disc or Belleville springs
6. Special purpose spring

### 1.1. LEAF SPRING

The leaf spring is main element of the suspension system. It can control for the wheels during acceleration, braking and turning, general movement caused by the road undulations. Leaf springs are designed in two methods: multi-leaf and mono leaf. The multi-leaf spring is made of several steel plates of different lengths stacked together. During normal operation, the spring compresses to absorb road shock. The leaf spring bends and slide on each other allowing suspension movement. An example of a mono-leaf spring is the tapered leaf spring. The leaf is thick in the middle and tapers towards the two ends. Many of these leaf springs are made of composite material, while others are made of steel. In most cases leaf springs are used in pairs mounted longitudinally (front and back). However, there is an increasing number of vehicle manufacturers using single transverse (side to side) mounted leaf spring.

Three types of leaf springs are:

1. Laminated or Multi-leaf springs.
2. Single or Mono-leaf springs.
3. Tapered leaf springs.

The third type of leaf spring is the combination of the above two. The multi-leaf springs are commonly used in the automobile suspension system at the rear side and are still in

use for commercial vehicles suspension system. It consists of a number of steel strips or leaves placed on top of each other and then clamped together. The type of application and load carried determines the length and number of leaves. The top leaf is called as the main leaf and the ends of the leaf are rolled to form the eye of the spring. This is for attachment to the vehicle chassis or body. The spring eye allows movement about the shackle and pin at the rear.

### II. MATERIALS FOR LEAF SPRINGS

Composite is usually made up of at least two materials out of which one is the binding material, also called matrix and the other is the reinforcement material. (fibre Kevlar and whiskers). The advantage of composite materials over conventional materials stem largely from their higher specific strength, stiffness, strong load carrying capacity and fatigue characteristics, which enables structural design to be more versatile.

Reinforcement provides strength and rigidity, helping to support structural load. The matrix or binder (organic or inorganic) maintains the position and orientation of the reinforcement. The reinforcement may be platelets, particles or fibres and are usually added to improve mechanical properties such as stiffness, strength and toughness of the matrix material. In present work Leaf spring is made up of Three materials like conventional steel, Coconut Fibre Reinforced Polyester Composite and Fibre reinforced polymer (S-glass/epoxy)

### 2.1. PROPERTIES OF THE MATERIAL

Table 1 Mechanical Properties of Steel

Property	value
Density	7850 kg/m <sup>3</sup>
Tensile Yield Strength	250 MPa
Poisson's ratio	0.3
Young's modulus	210 Gpa

**Table 2 Properties of Coir composite**

property	Value
Density	1380 kg/m <sup>3</sup>
Tensile Yield Strength	25 MPa.
Poisson's ratio	0.3
Young's modulus	315Gpa

**Table 3 Properties of S Glass/Epoxy composite**

property	Value
Density	2480 kg/m <sup>3</sup>
Tensile Yield Strength	4585 MPa.
Poisson's ratio	0.22
Young's modulus	86900Mpa



Figure: 2.1 Coir fibre



Figure: 2.2S Glass fibre

### III. DESIGN AND MODELING OF LEAF SPRING

#### 3.1. Model-Mahindra “Mahindra Jeep Commander 650 Di”



Leaf springs (also known as flat springs) are made out of flat plates. The advantage of leaf spring over helical spring is that the ends of the spring may be guided along a definite path as it deflects to act as a structural member in addition to energy absorbing device. Thus the leaf springs may carry lateral loads, brake torque; driving torque etc., in addition to shocks.

Let

t = Thickness of plate,

b = Width of plate,

2L = Effective length of leaf spring

l = ineffective length of leaf spring

N<sub>f</sub> = Number of full-length leaves

N<sub>g</sub> = Number of graduated leaves

n = Total number of leaves = N<sub>f</sub> + N<sub>g</sub>

2W = Central load acting

FOS = factor of safety

δf = Deflection in full length leaf

For reference and comparison sake a general leaf-spring of Model-Mahindra “Mahindra Jeep Commander 650 Di” was considered. The Dimensions of leaf spring as determined as follows.

Number of leaf springs = 10

Effective length of leaf spring = 1120 mm

Width of leaves = 50mm

Number of full-length leaves = 2

Number of graduated leaves = 8

Total number of leaves = 10

Central load acting = 2W = 1910Kg

$$2W = 1910 \times 10 \times 1.33 (\text{FOS})$$

$$= 25403 \text{ N}$$

$$2W = 25403/4$$

$$= 6350.7$$

$$W = 3200 \text{ N}$$

#### 3.2. Material used for leaf spring: structural steel

$$\begin{aligned} \text{Bending stress} &= \frac{6WL}{nb^2Z} \\ &= \frac{6 \times 1600 \times 560}{10 \times 50 \times 6^2} \\ &= 299 \text{ N/mm}^2 \end{aligned}$$

$$\begin{aligned} \text{Deflection in full length leaf} &= \frac{12WL^3}{Ebt^3(2nG+3nF)} \\ &= \frac{(12 \times 1600 \times 560^3)}{(207 \times (10)^3 \times 50 \times 6^3 (2 \times 8 + 3 \times 2))} \\ &= 68.5 \text{ mm.} \end{aligned}$$

Length of leaf

$$= \frac{\text{effective length}}{\text{number of leaves} - 1} + \text{ineffective length}$$

$$\text{Length of smallest leaf (leaf 1)} = \frac{1120}{10-1} + 90 = 214 \text{ mm}$$

$$\text{Length of second leaf} = \frac{1120}{10-1} \times 2 + 90 = 338 \text{ mm}$$

$$\text{Length of third leaf} = \frac{1120}{10-1} \times 3 + 90 = 463 \text{ mm}$$

$$\text{Length of fourth leaf} = \frac{1120}{10-1} \times 4 + 90 = 588 \text{ mm}$$

$$\text{Length of fifth leaf} = \frac{1120}{10-1} \times 5 + 90 = 712 \text{ mm}$$

$$\text{Length of sixth leaf} = \frac{1120}{10-1} \times 6 + 90 = 837 \text{ mm}$$

$$\text{Length of seventh leaf} = \frac{1120}{10-1} \times 7 + 90 = 961 \text{ mm}$$

$$\text{Length of eighth leaf} = \frac{1120}{10-1} \times 8 + 90 = 1085 \text{ mm}$$

$$\text{Length of ninth leaf} = 1120 \text{ mm}$$

$$\text{Length of tenth leaf} = 1120 \text{ mm}$$

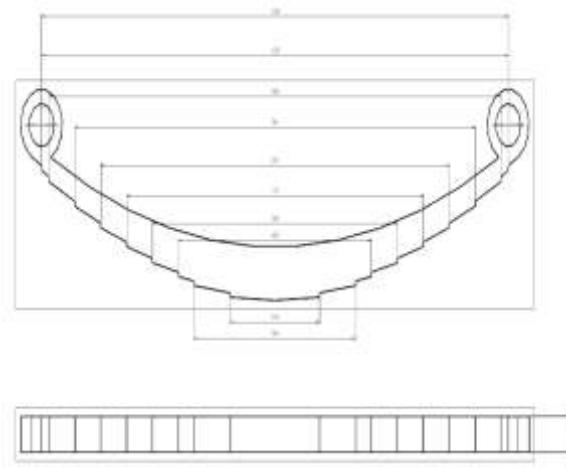


Fig:3.1 Drafting of steel leaf spring

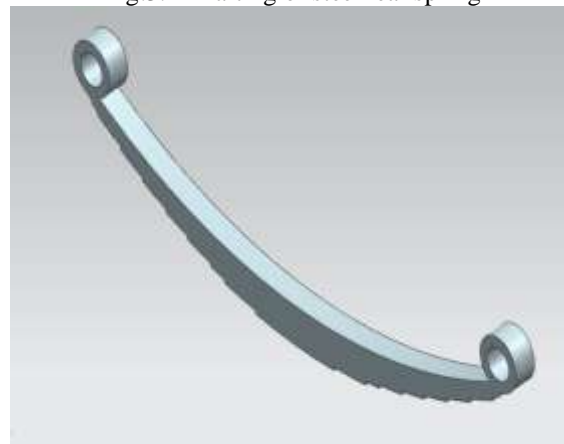


Figure: 3.2 CAD model of Steel leaf-spring.

### 3.3. Dimensions of Coconut Fibre Reinforced Polyester Composite Leaf-Spring and Fiber reinforced polymer (S-glass/epoxy):

The dimensions of composite leaf springs are set in such a way that, when 3200 N is applied, the deflection should be same as that of steel leaf-spring.

$$\text{Deflection in full length leaves} = \frac{12WL^3}{Eb^3(2nG+3nF)}$$

$$68.5 = \frac{(12 \times 1600 \times 560^3)}{(207 \times (10)^3 \times b \times 8^3 (2 \times 8 + 3 \times 2))}$$

$$b = 67 \text{ mm}$$

Firstly, length of 1420 mm and thickness of 8 mm is fixed and width is calculated using above formula.

$$\text{Length of leaf} = \frac{\text{effective length}}{\text{number of leaves} - 1} + \text{ineffective length}$$

$$\text{Length of smallest leaf (leaf 1)} = \frac{1420}{10-1} + 90 = 247.7 \text{ mm}$$

$$\text{Length of second leaf} = \frac{1420}{10-1} \times 2 + 90 = 405.55 \text{ mm}$$

$$\text{Length of third leaf} = \frac{1420}{10-1} \times 3 + 90 = 563.33 \text{ mm}$$

$$\text{Length of fourth leaf} = \frac{1420}{10-1} \times 4 + 90 = 721.11 \text{ mm}$$

$$\text{Length of fifth leaf} = \frac{1420}{10-1} \times 5 + 90 = 878.88 \text{ mm}$$

$$\text{Length of sixth leaf} = \frac{1420}{10-1} \times 6 + 90 = 1036.66 \text{ mm}$$

$$\text{Length of seventh leaf} = \frac{1420}{10-1} \times 7 + 90 = 1194.4 \text{ mm}$$

$$\text{Length of eighth leaf} = \frac{1420}{10-1} \times 8 + 90 = 1352.22 \text{ mm}$$

$$\text{Length of ninth leaf} = 1420 \text{ mm}$$

$$\text{Length of tenth leaf} = 1420 \text{ mm}$$

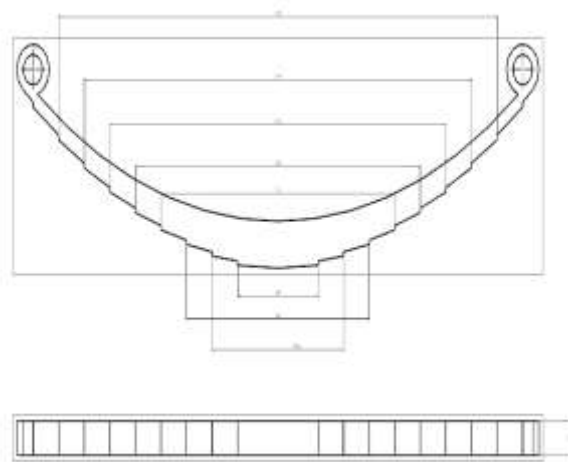


Fig:3.3 Drafting of coconut fibre and S Glass fibre composite leaf springs



Figure:3.4 CAD model of coconut fibre and S Glass fibre composite leaf springs

#### IV. ANALYSIS OF THE LEAF SPRING

ANSYS finite element analysis (FEA) is a computer based numerical technique for calculating the strength and behavior of engineering structures. It can be used to calculate deflection, stress, vibration, buckling behavior and many other phenomena. It can be used to analyze either small or large-scale deflection under loading or

applied displacement. It can analyze elastic deformation, or permanently bent out of shape plastic deformation. The computer is required because of the astronomical number of calculations needed to analyze a large structure. The power and low cost of modern computers has made Finite Element Analysis available to many disciplines and companies.



#### 4.1. Meshing:

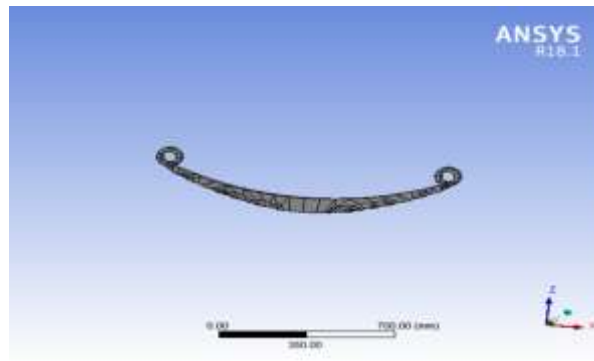


Fig 4.1: Mesh generation for steel leaf spring

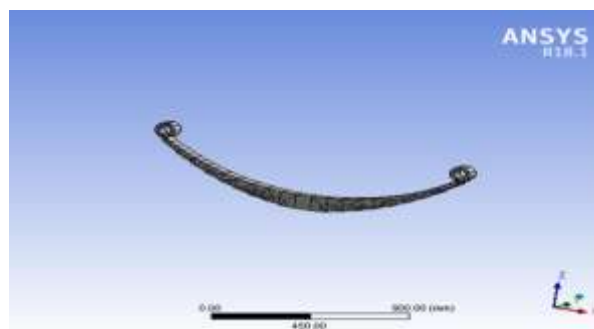


Fig 4.1: Mesh generation for coconut fibre composite leaf spring

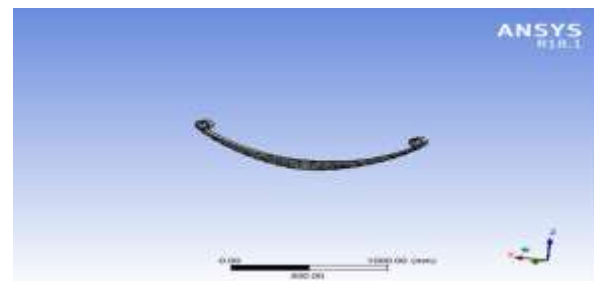
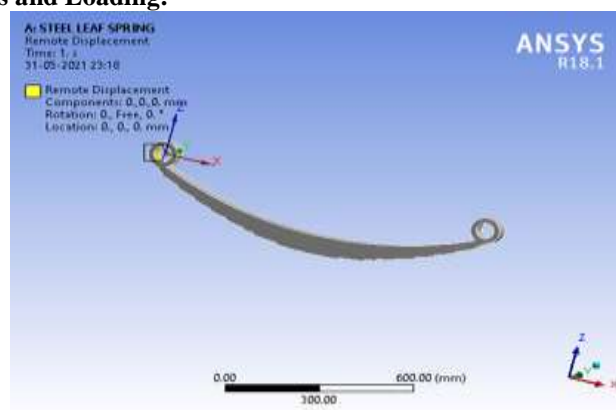


Fig 4.2: Mesh generation for S glass/Epoxy composite leaf spring

#### 4.2. Boundary Conditions and Loading:



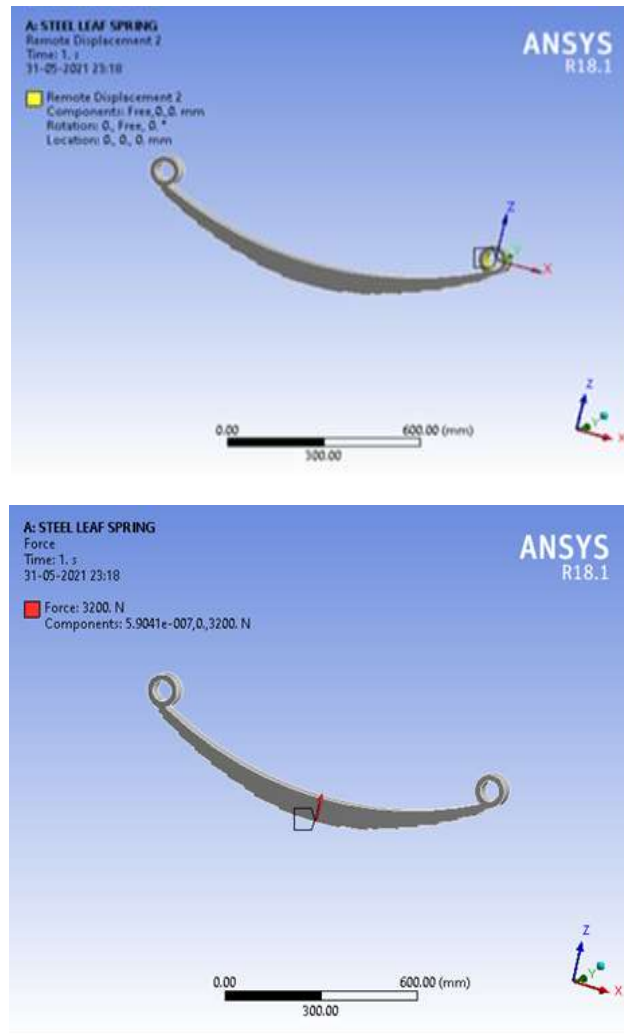


Fig 4.3: Boundary conditions for three leaf spring

## V. RESULTS AND DISCUSSIONS

The steel leaf-spring, coconut fibre composite leaf-spring and S Glass/Epoxy composite leaf-springs are analysed by giving constraints and the results obtained are as follows.

### 5.1 Analysis of steel Leaf Spring

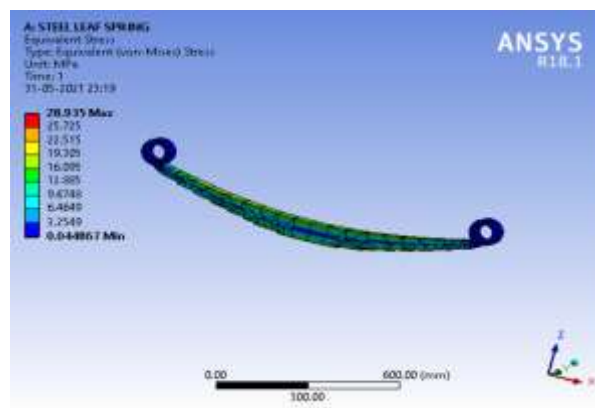


Fig.5.1.Max. Stress distribution



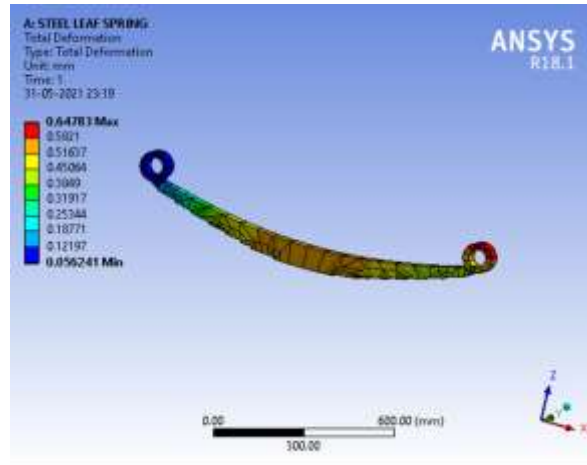


Fig.5.2.Max. Deformation

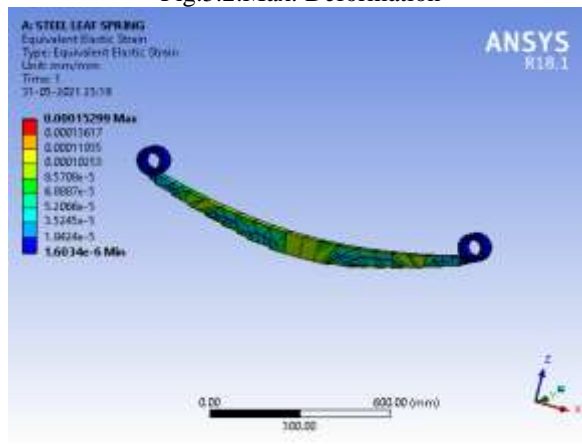
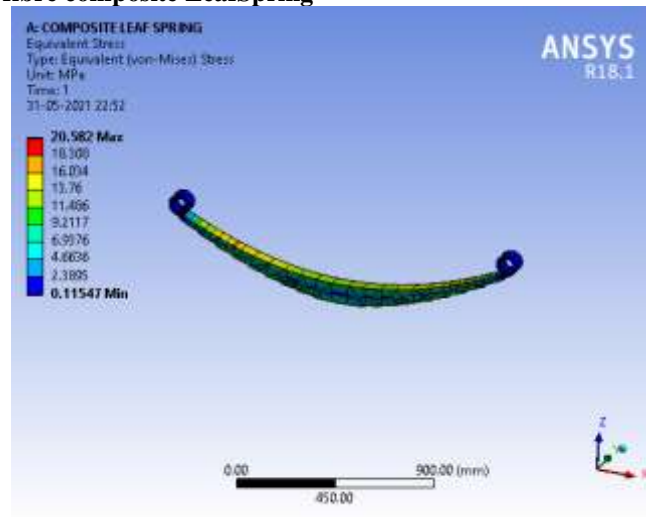


Fig.5.3.Max. strain distribution

## 5.2 Analysis of coconut fibre composite LeafSpring



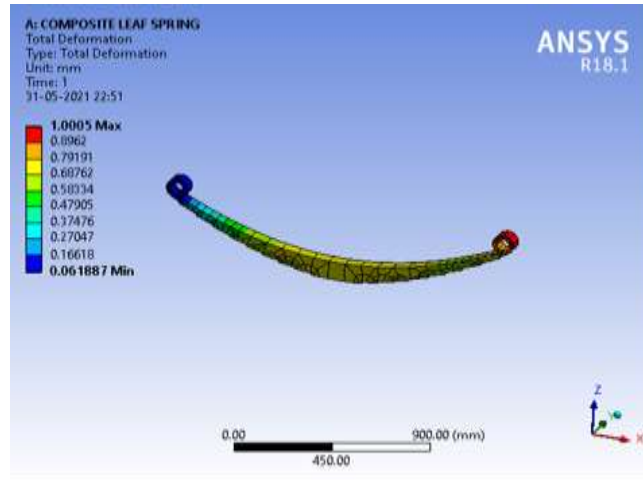


Fig.5.4. Max.Stressdistribution  
 Fig.5.5.Max. Deformation

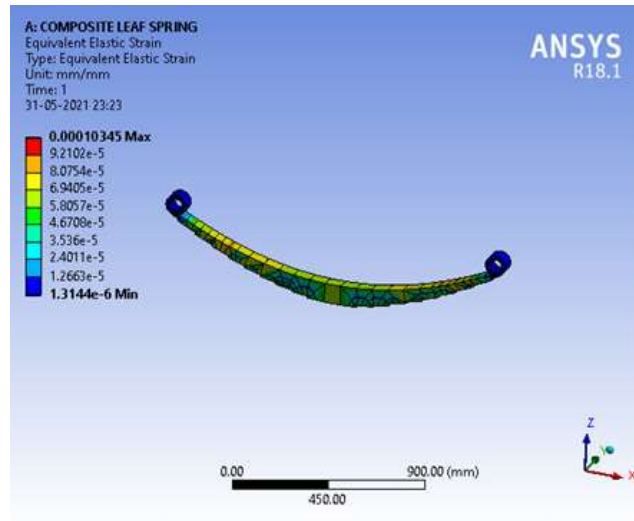


Fig.5.6.Max. strain distribution

### 5.3 Analysis of S GLASS/EPOXY composite Leaf Spring

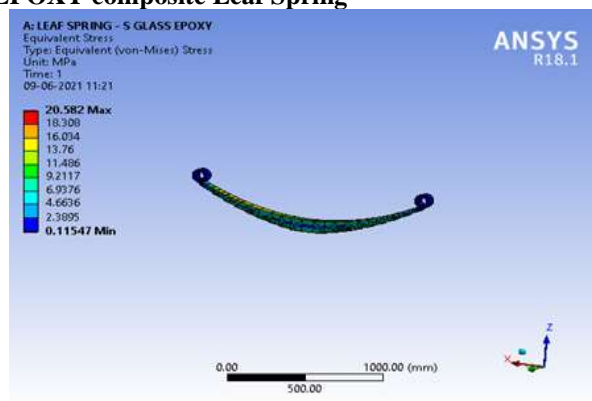


Fig.5.7.Max. Stress distribution

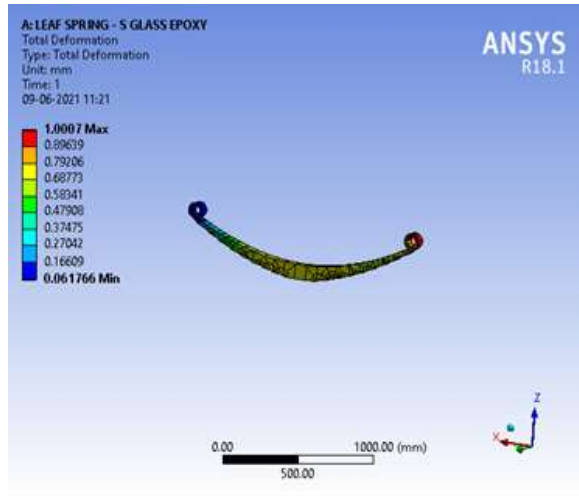


Fig.5.5.Max. Deformation

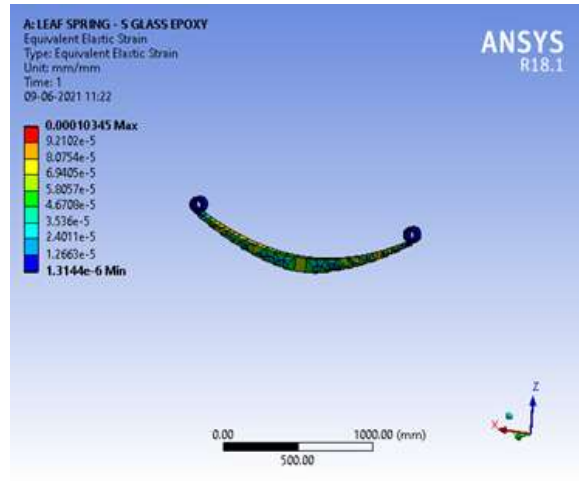


Fig.5.6.Max. strain distribution

Table 5.1. Comparison of Analysis Results of steel leaf and composite leaf springs

S . N o	Material for leaf spring	Max stress (Mpa)	Max Deform ation (mm)	Max. straindistribu tion	Weight (Kg)
1	Steel leaf spring	28.935	0.64783	0.00015299	<b>28.2727</b>
2	SGlass/ Epoxy composite leaf spring	20.582	1.0007	0.00010345	<b>17.9988</b>
3	Coconut fiber Composite leaf spring	20.582	1.0005	0.00010345	<b>10.0155</b>

Percentage of Weight saved (coconut fibrecomposite leaf spring) = 64.575%

$$= \frac{(28.2727 - 10.0155)}{28.2727}$$

Percentage of Weight saved (S Glass/Epoxy composite leaf spring)

$$= \frac{(28.2727 - 17.9988)}{28.2727}$$

$$= 36.338\%$$

From the results it can be observed that Equivalent stress generated in the Coconut Fiber Reinforced Polyester Composite leaf spring is less compared to S Glass/Epoxy composite leaf spring and steel leaf spring. Less maximum strain and acceptable deformation have been found in Coconut Fiber Reinforced Polyester Composite leaf spring compared to S Glass/Epoxy composite leaf spring and conventional steel leaf spring.

## VI. CONCLUSIONS

Under the same static load and deflection conditions, both composite and steel leaf springs show great difference in their weights. The weight of steel leaf spring is very high compared to that of composite leaf springs. The weight of steel spring is 28.2727 kgs whereas weight of S Glass/Epoxy composite leaf spring is 17.9988Kg and weight of coconut fibre composite leaf spring is 10.0155 kgs. The induced stress of coconut fibre composite leaf spring is 20.582 MPa which is equal to the S Glass/Epoxy composite leaf spring less than that of steel leaf spring 28.935 MPa. The results obtained for deflection of steel leaf spring and composite leaf springs are in acceptable range. Composite leaf spring can be used in light weight vehicles, where weight is an important factor, whereas steel spring can be used in budget cars for its low cost of manufacturing.

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